

Hybrid PV-Geo-Aerothermal System for Self-Sufficient Energy Buildings

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Abstract

This article analyzes the feasibility of a self-sufficient domestic energy installation powered by a hybrid renewable energy system consisting of a photovoltaic array and a domestic geothermal unit, supported by a battery bank to compensate for energy imbalances during periods of supply deficit. Energy consumption includes an aerothermal heat pump for heating, air conditioning, and domestic hot water, as well as appliances and accessories for the house's current operation. The analysis includes experimental validation of the self-sufficient system, considering the overall annual energy balance. The study analyzes the daily evolution of energy consumption and supply to determine the distribution of the energy balance over a shorter period and obtain a more precise analysis. The experimental results show that the designed system achieves a zero-energy balance with an accuracy of 0.02% for the overall annual energy balance. The monthly evaluation of the daily energy balance verifies that the selected battery bank compensates for the daily energy imbalance between supply and demand. The system design demonstrates the feasibility of the installation, provided the dwelling has sufficient space for the photovoltaic array and the geothermal unit. In the case studied, a 75 m² roof and a 50 m² plot of land provide enough space for a 26-panel photovoltaic array of 425 W each and a 3-kW geothermal unit for thermal energy supply.

Keywords: Building Energy Self-Sufficiency, Hybrid Renewable Energy System, Annual Net Zero Energy Balance

1. Introduction

Currently, many efforts focus on achieving energy self-sufficiency in buildings by using renewable energies as the primary source of energy supply [1-3]. The goals are to reduce energy consumption and dependence on fossil fuels, while minimizing the environmental impact of energy use for building air conditioning. The modern trend uses hybrid systems instead of single renewable sources since the hybridization supplies energy more continuously and improves the system efficiency [4-6]. Common hybridization systems in buildings include solar photovoltaic arrays, domestic wind turbines, heat pumps, electric thermal storage, and hydropower units [7-10]. The hybridization benefits from complementary energy sources, improving global efficiency and providing a reliable power supply [11-15].

Householders play a crucial role in advancing energy sufficiency since they represent a significant percentage of energy consumption in modern society. Nevertheless, their behavior depends not only on personal motivations but also on external factors such as policy mixes, specific technology availability, and economic profit [16-20]. Two opposing factors merge into the householder's decision to install a renewable energy hybrid system: the installation cost and the money saved in big cities, where power supply is produced at relatively low cost, the investment in a renewable energy system is a barrier to installing household self-sufficient energy systems [21-26]. On the other hand, the prospect of having energy at zero cost represents an attractive factor in favor of renewable energy in domestic installations [27-31].

Given the trade-off between a costly investment and an energy supply system with only maintenance costs, it is challenging to decide which system best suits a specific home. There is no single answer, as each house has unique characteristics and energy needs based on the residents' lifestyles [32-35]. Several proposals exist for achieving a sustainable and efficient household with zero external energy

dependence. These include single or hybrid systems, PV arrays with battery blocks solar, thermal systems with storage, domestic micro wind turbines solar thermal, and photovoltaic panels, PV systems connected heat pumps, domestic biomass devices and micro geothermal systems with PV arrays heat pump units or small wind turbines [36-56].

The above list shows the current state of the art; nevertheless, alternative solutions arise for supplying heat and electricity to detached and semi-detached houses. In this paper, we propose a hybrid system that combines a photovoltaic array for electric power supply with an aerothermal unit for sanitary water, heating, and air conditioning, and a geothermal system acting as a heat reservoir and complementary power source, heat, and electricity, when the PV array does not cover the house's energy requirements. The proposed power source combination benefits from a versatile power supply, the photovoltaic-aerothermal hybrid system, and an additional stable power source, the geothermal system, that stores energy for energy compensation when necessary.

2. Power Sources and Energy Requirements

The energy sources in a household come from heat and electricity for sanitary water, heating and air conditioning, and power supply for domestic appliances. In an energy self-sufficient system, the photovoltaic array supplies electricity for household lighting and electric appliances. Heating and air conditioning may operate on an electric current basis using a heat pump or a conventional gas boiler. The heating and air conditioning system may use wall radiators, underfloor heating, or fan-coil units [57-62]. Since the PV source does not operate during night and cloudy periods, the installation requires an electric storage system, currently a battery block. Thermal supply requires a storage unit because the aerothermal system operates under a photovoltaic array basis; therefore, when there is no solar radiation, the electric current from the PV power supply stops, and the aerothermal unit cannot continue working, interrupting the heat power supply for sanitary water, heating, and air conditioning. We can store thermal energy in several forms [63-64]; however, the typical form in a house is a hot water tank that preserves thermal energy for a short period. This situation generates a frequent heat power supply to compensate for thermal losses [65-66]. The thermal energy replacement proceeds from the aerothermal unit, which activates a heat exchange process to maintain the hot water tank temperature within limits.

An alternative solution for the hot water tank is to store thermal energy in the ground, as its temperature variations are low throughout the entire year, thereby minimizing the heat power required to maintain a constant temperature in the thermal reservoir. We, therefore, propose a ground thermal reservoir using a buried heat exchanger connected to the house's sanitary water circuit. In this way, the ground acts as a heat power supply when the aerothermal unit is not operational due to a PV power supply interruption. Another solution is a gravel-water pit which preserves thermal energy longer than a single water tank due to the gravel's higher heat capacity. In both cases, a ground reservoir or gravel-water pit, the sanitary water system requires an intermediate heat pump to raise the temperature from the thermal reservoir to the sanitary water circuit [67-72].

3. System Design and Characteristics

The proposed system consists of a set of 26 photovoltaic panels, lying on the tilted roof of a semi-detached house. Since the roof is oriented East and West, we divided the PV array into two groups of 13 panels for each orientation. Each panel has a peak power of 425 watts, resulting in an equivalent power of 8.288 kW for a south-oriented photovoltaic array with the same inclination as mentioned. Table 1 shows the technical characteristics of the PV panels.

| | |
|-------------------------------------|---------|
| Nominal Maximum Power (P_{max}) | 425 W |
| Operating Voltage (V_{mp}) | 38.25 V |
| Operating Current (I_{mp}) | 11.11 A |
| Open Circuit Voltage (V_{oc}) | 45.64 V |
| Short Circuit Current (I_{sc}) | 11.82 A |
| Module Efficiency | 20.6% |

Table 1: PV panel technical characteristics

The two strings of the PV array are connected to a 8.6 kW system with a double MPPT unit, one for every string. The input voltage is 500 VDC per string, and the maximum current is 11.11 A. The inverter is an island-type, operating off-grid for household energy self-sufficiency. The inverter includes a control module allowing the unit to connect to the house electric circuit, the battery block, the aerothermal unit, and the ground thermal heat pump. Table 2 shows the inverter technical characteristics.

| | |
|-------------------------------|-----------|
| DC Input data | |
| Maximum operating power | 8600 W |
| Maximum power per MPPT/string | 4300 W |
| Maximum voltage | 1000 V |
| Activation voltage | 170 V |
| Nominal operating voltage | 500 V |
| AC Output data | |
| Maximum operating power | 8600 W |
| Maximum current | 37.4 A |
| Operating voltage | 230 V |
| Operating voltage range | 180-276 V |
| Performance data | |
| Power factor | 1 |
| Maximum efficiency | 98% |
| Average efficiency | 97.5% |
| MPPT efficiency | >99.9% |
| Power losses | < 1W |

Table 2: Inverter technical characteristics

The battery block is a four 6.0 kW unit operating at 48 VDC nominal voltage, with a working range of 42-54 VDC [73]. The maximum delivered energy is 5.4 kW per unit, with a 10% reservoir for security reasons. The recommended and maximum discharge currents are 50 and 75 A. The battery charge is protected by a BMS, which controls voltage, current, and temperature of every battery cell. Table 3 shows the battery block characteristics.

| | |
|-----------------------------|------------------------|
| Material | Lithium-Ferrophosphate |
| Nominal Voltage | 48 V |
| Nominal Capacity | 6.0 kW |
| Available Capacity | 5.4 kW |
| Charge voltage | 53.2 V |
| Working voltage range | 42-54 V |
| Operating discharge current | 62.5 A |
| Max. discharge current | 93.75 A |
| Communication | RS232, RS482, CAN |
| Number of units | 4 |
| Working temp. range | 0~55 °C (charge) |
| | -20~55 °C (discharge) |
| | -10~35 °C (storage) |
| Lifespan | >6000 cycles |

Table 3: Battery block technical characteristics

Figures 1 and 2 show the schematic representation of the system design.

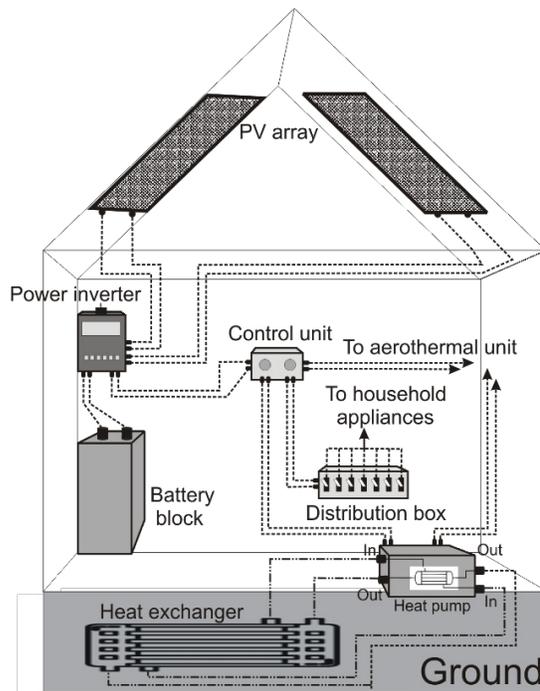


Figure 1: Layout of the electric distribution and the underground heat exchanger

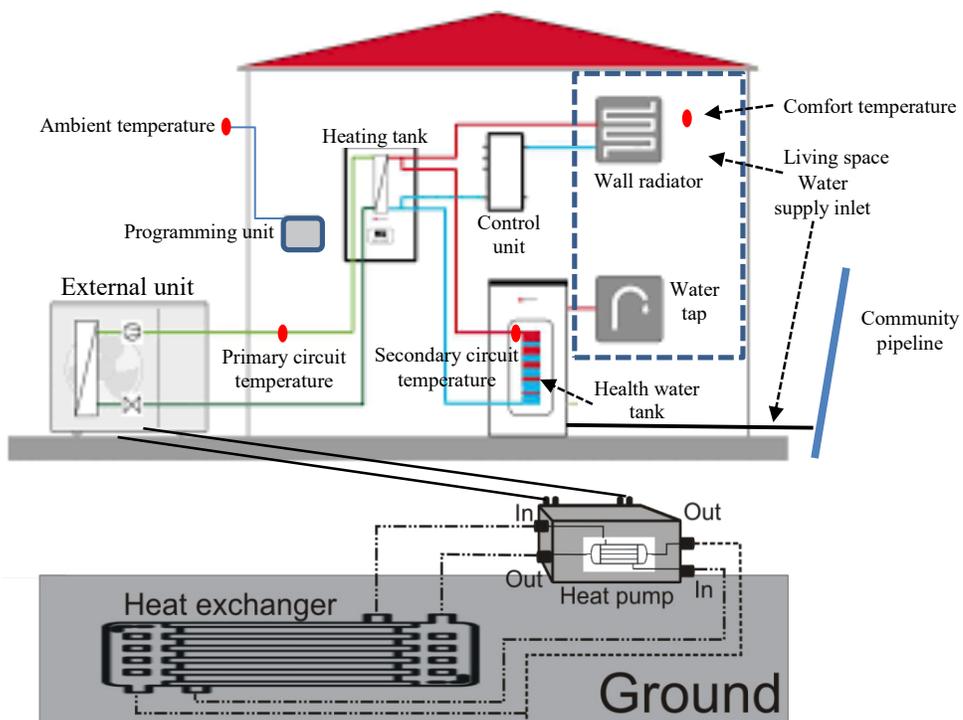


Figure 2: Layout of the thermal energy distribution (aerothermal unit and underground exchanger with assisting heat pump)

The aerothermal system is an air-water heat pump of 6 kW maximum power. The control unit that manages the aerothermal operation allows heating curve selection to increase or decrease the raising of water temperature. The heating curve selection responds to a PID control process [74]. The control unit also allows regulating hot water tank, and heating and air conditioning temperatures. We control the temperature regulation automatically through a home automation process assisted by Artificial Intelligence to optimize energy consumption [75]. Table 4 shows the aerothermal technical characteristics.

| | |
|--|---------------|
| Electrical data | |
| Maximum operating power | 6.0 kW |
| Power factor | 1.0 |
| Maximum operating current | 23.3 A |
| Fan power consumption | 160 W |
| Heat pump operating power | 3-87 W |
| Thermal data | |
| Sanitary water temperature range | 20-75 °C |
| Water operating pressure range | 0.05-0.30 MPa |
| Water flow range | 995-2065 L/h |
| Heating flow range | 45-150 L/h |
| Power supply in heating mode (primary circuit) | 5.70 kW |
| Power supply in cooling mode (primary circuit) | 7.80 kW |
| Power supply in heating mode (secondary circuit) | 5.40-15.00 kW |
| Power supply in cooling mode (secondary circuit) | 4.30-12.00 kW |
| Heating COP | 4.20 |
| Cooling COP | 4.60 |
| Effective power consumption (heating mode) | 1.36 kW |
| Effective power consumption (cooling mode) | 2.35 kW |

Table 4: Aerothermal unit technical characteristics

The underground environment operates at variable temperatures depending on the season of the year; however, this variation is low, according to data from the geographical region where the house is located. Previous work shows that the ground temperature profile varies in the 19 °C to 25 °C range, with the minimum temperature corresponding to winter and the maximum to summer [76]; therefore, we can operate at an average temperature of 22 °C with a maximum temperature oscillation of ±3 °C. Since sanitary water operates in the 45°C to 60°C range, it is necessary to increase the underground reservoir temperature through a micro heat pump (Figure 1), whose principal goal is to maintain the health water tank at constant temperature, minimizing the dependence on the aerothermal unit, and improving the system's global efficiency. Because the temperature variation in the health water tank only occurs during sanitary water disposal, and this temperature variation is relatively low, the energy requirement to maintain constant the temperature of the health water tank is moderate; therefore, the micro heat pump power consumption is low, especially compared with the energy supply if the aerothermal unit is responsible to maintain the health water tank temperature constant. Table 5 shows the geothermal heat pump technical characteristics.

| | |
|--------------------------------|---------------------|
| Available land area | 71.5 m ² |
| Underground system area | 50 m ² |
| Underground system depth | 1.5 m |
| Heat pump power range | 0.19-0.76 kW |
| Average heat pump operating | 0.46 kW |
| Heating COP | 3.5 |
| Cooling COP | 4.0 |
| Maximum heating power supply | 3.04 kW |
| Maximum cooling heating supply | 1.86 kW |

Table 5: Geothermal heat pump technical characteristics

3.Theoretical Background

In a standard household, we have two types of energy requirements: electric and thermal. Since electric power supply comes directly from the photovoltaic array, and the thermal energy proceeds from the aerothermal unit, and because the PV array and aerothermal unit efficiencies are different, we can express the household electric (P_{el}) and thermal (P_{th}) power requirement as in the following equations:

$$P_{el,c} = \frac{\xi_{el} \eta_{inv} + \xi_{el}^L}{t_{op}} \quad (1)$$

$$P_{th,c} = \frac{\xi_{th} + \xi_{th}^L}{t_{op} COP} \quad (2)$$

ξ_{el} and ξ_{th} correspond to the electric and thermal household energy requirements, η_{inv} is the inverter efficiency, ξ_{el}^L and ξ_{th}^L are the electric and thermal losses, COP is the aerothermal unit coefficient of performance, and t_{op} is the operation time. The subscript c accounts for consumption.

The electric energy requirements depend on the individual appliance power, its working time, and the number of elements, as in equation 3:

$$\xi_{el,c} = \sum_{i=1}^n \left(\sum_{j=1}^m P_j t_j \right)_i \quad (3)$$

P_j is the individual appliance power, and t_j is the working time. Subscript i accounts for the number of elements of any individual appliance of the same type, j .

On the other hand, thermal energy requirements include sanitary water and heating or air conditioning. To clarify, the thermal energy needs can be represented as shown in equation 4:

$$\xi_{th,c} = m_{SW} c_w \Delta T_{SW} + m_{air} c_{air} \Delta T_{air} \quad (4)$$

Electric and thermal losses derive from the classical expressions [77-78]:

$$\xi_{el}^L = \sum_k I_k R_k ; \xi_{th}^L = U_L S \Delta T_{room} \quad (5)$$

I and R are the current and electrical resistance of every wire in the electric installation, and k is the number of electric wires. U_L is the household global thermal losses coefficient, S is the household wall and roof surface, and ΔT_{room} is the temperature difference between the household interior and the environment.

Replacing Equations 3, 4, and 5 in Equations 1 and 2, we have:

$$P_{el,c} = \frac{1}{t_{op}} \left[\sum_{i=1}^n \left(\sum_{j=1}^m P_j t_j \right)_i \eta_{inv} + \sum_k I_k R_k \right] \quad (6)$$

$$P_{th,c} = \frac{1}{t_{op}} \left[(m_{SW} c_w \Delta T_{SW} + m_{air} c_{air} \Delta T_{air}) + U_L S \Delta T_{room} \right] \quad (7)$$

Converting power into energy:

$$\xi_{el,c} = \left[\sum_{i=1}^n \left(\sum_{j=1}^m P_j t_j \right)_i \eta_{inv} + \sum_k I_k R_k \right] \quad (8)$$

$$\xi_{th,c} = m_{SW} c_w \Delta T_{SW} + m_{air} c_{air} \Delta T_{air} + U_L S \Delta T_{room} \quad (9)$$

Electric power is supplied by the photovoltaic array and the battery block. The PV system supplies energy as a function of the solar radiation level, which is variable throughout the day; therefore:

$$P_{el,s}^{PV} = f_{or} N_{PV} \frac{G}{1000} (PV)_M \quad (10)$$

$(PV)_M$ is the photovoltaic panel peak power, G is the solar radiation, N_{PV} is the number of photovoltaic panels, and f_{or} is the correction factor due to the East-West orientation of the PV array. The superscript PV indicates that the power supply corresponds to for the photovoltaic array.

Converting power into energy, we have:

$$\xi_{el,s}^{PV} = f_{or} N_{PV} \frac{(PV)_M}{1000} \sum_l G_l \quad (11)$$

The subscript l corresponds to the operating time interval, and G_l is the solar radiation corresponding to this interval.

The time interval depends on the measuring protocol and accuracy of the data recording system; this time interval can be as low as a fraction of a second. However, in current practice, to avoid excessive data volume, the solar radiation is averaged over an hour; therefore, Equation 11 converts into:

$$\xi_{el,s}^{PV} = f_{or} N_{PV} \frac{(PV)_M}{1000} \sum_{t=1}^{24} G_t \quad (12)$$

The subscript t corresponds to the daily hour interval, and G_t is the average solar radiation over this hour. The subscript s indicates energy supply.

Thermal energy proceeds from the aerothermal unit and the underground reservoir. In the case of the aerothermal unit, we can express the thermal energy as:

$$\xi_{th,s}^{aero} = \xi_{th,c} = \xi_{el,s}^{aero} (COP)_{aero} \quad (13)$$

$\xi_{el,s}^{aero}$ is the fraction of electric energy generated by the PV array to produce thermal energy in the aerothermal unit heat pump. $(COP)_{aero}$ is the aerothermal unit coefficient of performance for heating or air conditioning.

If we convert thermal energy into electrical at the aerothermal unit, replacing Equation 9 in Equation 13, we obtain:

$$\xi_{el,s}^{aero} = \frac{m_{SW} c_w \Delta T_{SW} + m_{air} c_{air} \Delta T_{air} + U_L S \Delta T_{room}}{(COP)_{aero}} \quad (14)$$

Because we use the geothermal reservoir to replace thermal energy in the healthy water tank, Equation 14 should be expressed in the following way:

$$\xi_{el,s}^{aero} = \frac{m_{air} c_{air} \Delta T_{air} + U_L S \Delta T_{room}}{(COP)_{aero}} \quad (15)$$

The supplied energy by the geothermal heat pump is, therefore, given by:

$$\xi_{el,s}^{geo} = \frac{m_{SW} c_w \Delta T_{SW}}{(COP)_{geo}} \quad (16)$$

We can obtain the electric energy balance by comparing the power supply from the PV array and the battery block with the household, aerothermal and geothermal heat pump electric energy consumption.

The battery block health is critical since it represents the backup system that guarantees the household operational viability in case the PV array does not supply enough power to cover the household energy demand. If the electric energy balance is negative, the batteries block discharges, supplying energy to compensate for the energy deficit. If positive, the batteries block charges. Therefore, the battery output/input energy is given by:

a) Battery discharge process

$$\xi_{el,s}^{bat} \Big|_d = \left[\sum_{i=1}^n \left(\sum_{j=1}^m P_j t_j \right)_i \eta_{inv} + \sum_k I_k R_k \right] + \frac{m_{air} c_{air} \Delta T_{air}}{(COP)_{aero}} + \frac{+U_L S \Delta T_{room}}{(COP)_{geo}} - \xi_{el,s}^{PV} \quad (17)$$

b) Battery charge process

$$\xi_{el,s}^{bat} \Big|_{ch} = \xi_{el,s}^{PV} - \left[\sum_{i=1}^n \left(\sum_{j=1}^m P_j t_j \right) \eta_{inv} + \sum_k I_k R_k \right] - \frac{m_{air} c_{air} \Delta T_{air}}{(COP)_{aero}} - \frac{U_L S \Delta T_{room}}{(COP)_{geo}} \quad (18)$$

In terms of electric parameters:

a) Battery discharge process

$$\xi_{el,s}^{bat} \Big|_h = V_D I_D t_D \quad (19)$$

b) Battery charge process

$$\xi_{el,s}^{bat} \Big|_h = V_{ch} I_{ch} t_{ch} \quad (20)$$

V is the battery voltage, I is the current, and t is the operating time. Subscripts D and ch account for discharge and charge processes.

During the battery charge or discharge process, the remaining energy increases or decreases, modifying the state of charge of the battery (SOC) according to:

$$SOC = 1 - DOD = 1 - \sum_{y=1}^z \frac{(I_D t_D)_y}{C_{bat}} \quad (21)$$

For the discharge process, and:

$$SOC = \sum_{y=1}^z \frac{(I_{ch} t_{ch})_y}{C_{bat}} \quad (22)$$

For the charge process.

C_{bat} accounts for the battery block capacity, and subscript z for the number of charge/discharge processes. Since the battery capacity changes with discharge current, Equations 21 and 22 convert into:

$$(SOC)_D = 1 - \sum_{y=1}^z \frac{(I_D t_D)_y}{C_n f_C} \quad (23)$$

$$(SOC)_{ch} = \sum_{y=1}^z \frac{(I_{ch} t_{ch})_y}{C_n f_C} \quad (24)$$

f_C represents the battery capacity correction factor, given by [79]:

$$f_C = 0.9541 (t_D)^{0.0148} = 0.9541 \left(\frac{I_D}{C_n} \right)^{0.0148} \quad (25)$$

C_n is the battery nominal capacity, corresponding to the delivered charge in the standard discharge time.

Replacing Equation 25 in Equations 23 and 24:

$$(SOC)_D = 1 - \frac{1}{0.9541} \sum_{y=1}^z \left[\left(\frac{I_D}{C_n} \right)^{1.9852} \right]_y \quad (26)$$

$$(SOC)_{ch} = \frac{1}{0.9541} \sum_{y=1}^z \left[\left(\frac{I_D}{C_n} \right)^{1.9852} \right] \quad (27)$$

Applying the Ohm's law:

$$(SOC)_D = 1 - \frac{1}{0.9541} \sum_{y=1}^z \left[\left(\frac{1}{C_n} \right)^{1.9852} \left(\frac{\xi_{bat}}{V_{bat}} C_n \right)^{1.9852/2} \right] = 1 - \frac{1}{0.9541} \sum_{y=1}^z \left[(C_n)^{-0.9926} \left(\frac{\xi_{bat}}{V_{bat}} \right)^{0.9926} \right] \quad (28)$$

$$(SOC)_{ch} = \frac{1}{0.9541} \sum_{y=1}^z \left[\left(\frac{I_D}{C_n} \right)^{1.9852} \right] = \frac{1}{0.9541} \sum_{y=1}^z \left[(C_n)^{-0.9926} \left(\frac{\xi_{bat}}{V_{bat}} \right)^{0.9926} \right] \quad (29)$$

Now, replacing Equations 17 and 18 in Equations 28 and 29:

$$(SOC)_D = 1 - \frac{1}{0.9541} \sum_{y=1}^z \left[(C_n)^{-0.9926} \left(\frac{\left[\sum_{i=1}^n \left(\sum_{j=1}^m P_j t_j \right)_i \eta_{inv} + \sum_k I_k R_k \right] + \frac{m_{air} c_{air} \Delta T_{air}}{(COP)_{aero}} + \frac{+U_L S \Delta T_{room}}{(COP)_{geo}} - \xi_{el,s}^{PV}}{V_{bat}} \right)^{0.9926} \right] \quad (30)$$

$$(SOC)_{ch} = \frac{1}{0.9541} \sum_{y=1}^z \left[(C_n)^{-0.9926} \left(\frac{\xi_{el,s}^{PV} - \left[\sum_{i=1}^n \left(\sum_{j=1}^m P_j t_j \right)_i \eta_{inv} + \sum_k I_k R_k \right] - \frac{m_{air} c_{air} \Delta T_{air}}{(COP)_{aero}} - \frac{U_L S \Delta T_{room}}{(COP)_{geo}}}{V_{bat}} \right)^{0.9926} \right] \quad (31)$$

We achieve the household energy sustainability if the battery state of charge remains positive, especially during the discharge process; therefore, the condition imposed is $(SOC)_D > 0$, where the $(SOC)_D$ value derives from Equation 30.

The battery state of charge during charge is also important because it defines the starting point for the discharge process.

4. Experimental Tests and Results

We tested the household energy sustainability through experimental tests run for an entire year to compensate seasonal influence on the energy consumption. Table 6 shows the testing parameters of the experimental system.

| | |
|----------------------------------|---|
| Energy system | Photovoltaic |
| Subsystem | Parameter |
| PV array (strings 1 and 2) | Output voltage |
| | Output current |
| Inverter | Input DC power (string 1, 2, and total) |
| | Output AC voltage |
| | Output AC current |
| | Global output power |
| Power factor | |
| Energy system | Aerothermal |
| Subsystem | Parameter |
| Health water tank (HWT) | Input temperature (from network) |
| | Output temperature (to sanitary circuit) |
| Reservoir tank (RT) | Input temperature (from HWT) |
| | Output temperature (to HWT) |
| | Input temperature (from aerothermal unit) |
| | Output temperature (to aerothermal unit) |
| Aerothermal unit | Working pressure |
| | Water flow (primary circuit) |
| | Water flow (sanitary water circuit) |
| | Air temperature |
| | Electric energy consumption |
| | Thermal energy generation |
| Coefficient of Performance (COP) | |
| Energy system | Battery block |
| Subsystem | Parameter |
| Battery | Operational voltage |
| | Input-Output Current |
| | Input-Output charge |
| | Delivered power |
| | State of Charge |
| Energy system | Thermal storage System |
| Subsystem | Parameter |
| Heat pump | Working pressure |
| | Water flow (primary circuit) |
| | Water flow (sanitary water circuit) |
| | Air temperature |
| | Electric energy consumption |
| | Thermal energy generation |
| | Coefficient of Performance (COP) |

Table 6: Experimental system testing parameters

Solar radiation measurements are collected by a SKS 1110 pyranometer sensor [80], which has a linearity error of less than 0.2%, current sensitivity of $50 \mu\text{A}/\text{kW}\cdot\text{m}^{-2}$, voltage sensitivity of $10 \text{ mV}/\text{kW}\cdot\text{m}^{-2}$, and operational range of $0\text{-}5000 \text{ W}/\text{m}^2$. The response sensor time of 10 ns, assuring an immediate detection of any solar radiation variation.

The PV inverter is a chain monophasic unit, Greenheiss Series GH-IT [81], measuring the input voltage and current from the two PV array strings and the output voltage and current for the household electric circuit. The internal unit evaluates the power factor. The power factor has a 98.2% accuracy. The built-in energy counter operates in the range $0\text{-}9999 \text{ kWh}$, with a resolution of 0.01% and an accuracy

of 98.5%.

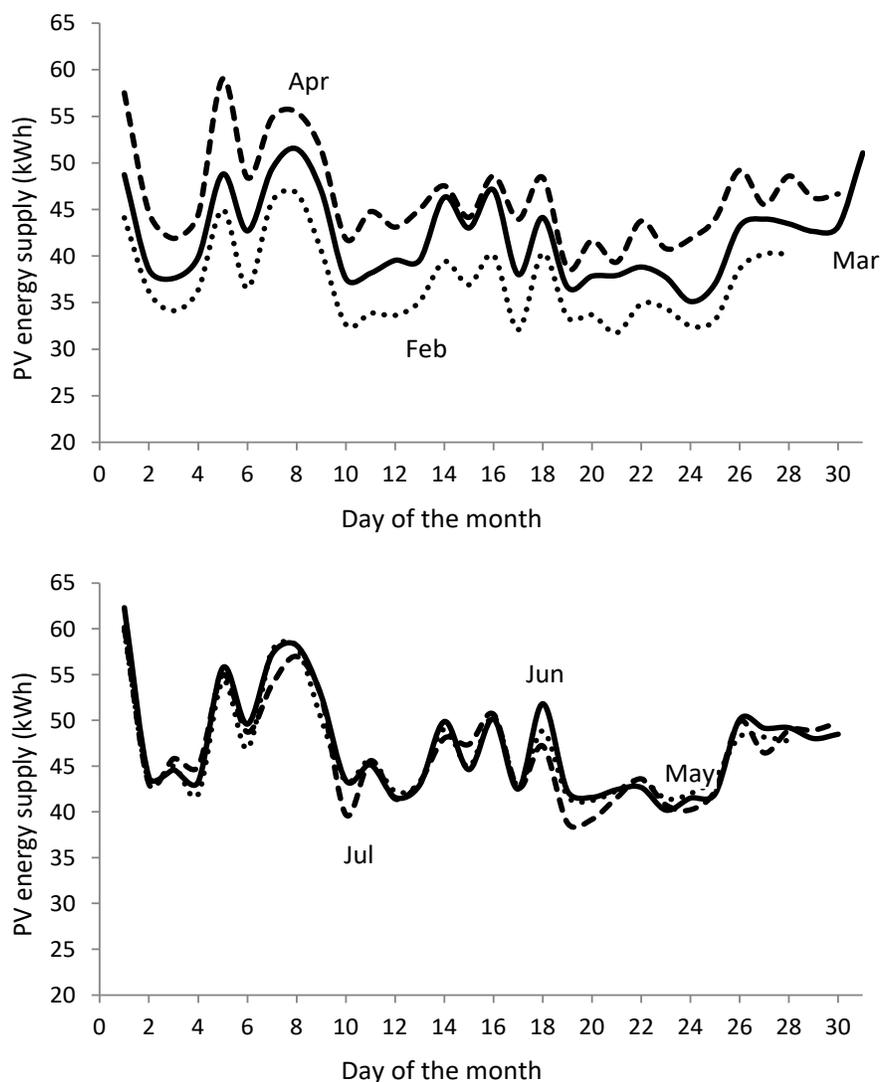
Battery and PV array electric parameters are measured using a high-resolution power analyzer, the PCE-PA-6000 [82], which has an accuracy of 0.1V and 10 mA for voltage and current detection, respectively. The power analyzer calculates the input and output power to and from the battery with an accuracy of 0.1 W for a measuring range below 1 kW, and of 1.0 W for a measuring range above 1 kW.

The aerothermal unit, Saunier Duval GeniaAir HA 15-6 O B3 [83], uses a high-resolution pressure gauge of 1Pa sensitivity to measure the working pressure, operating in the 0-2 MPa range. Temperature sensors are of the T-type, class 1, having a resolution of 0.5° C, and an operational range of -185 to 300° C. temperatures are continuously controlled through a multi-channel PicoLog data acquisition logger [84] connected to the control unit.

The geothermal heat pump, ecoForest model ecoGEO+AU6 [85], has similar characteristics to the aerothermal unit. Tests are run for one year, measuring values every minute and averaging the data hourly to match standard operating procedures. The control unit collects and records data, sending results to a connected computer for evaluation.

Showing hourly values for a whole year involves too much data. Instead, we display the average value of every day for each month.

The first group of tests results corresponds to the evolution of the photovoltaic energy supply (Figure 3). Every figure corresponds to a group of three months, corresponding to a yearly period centered on the solar solstice and equinox; February to April for the spring equinox, May to July for the summer solstice, August to October for the autumn equinox, and November to January for the winter solstice.



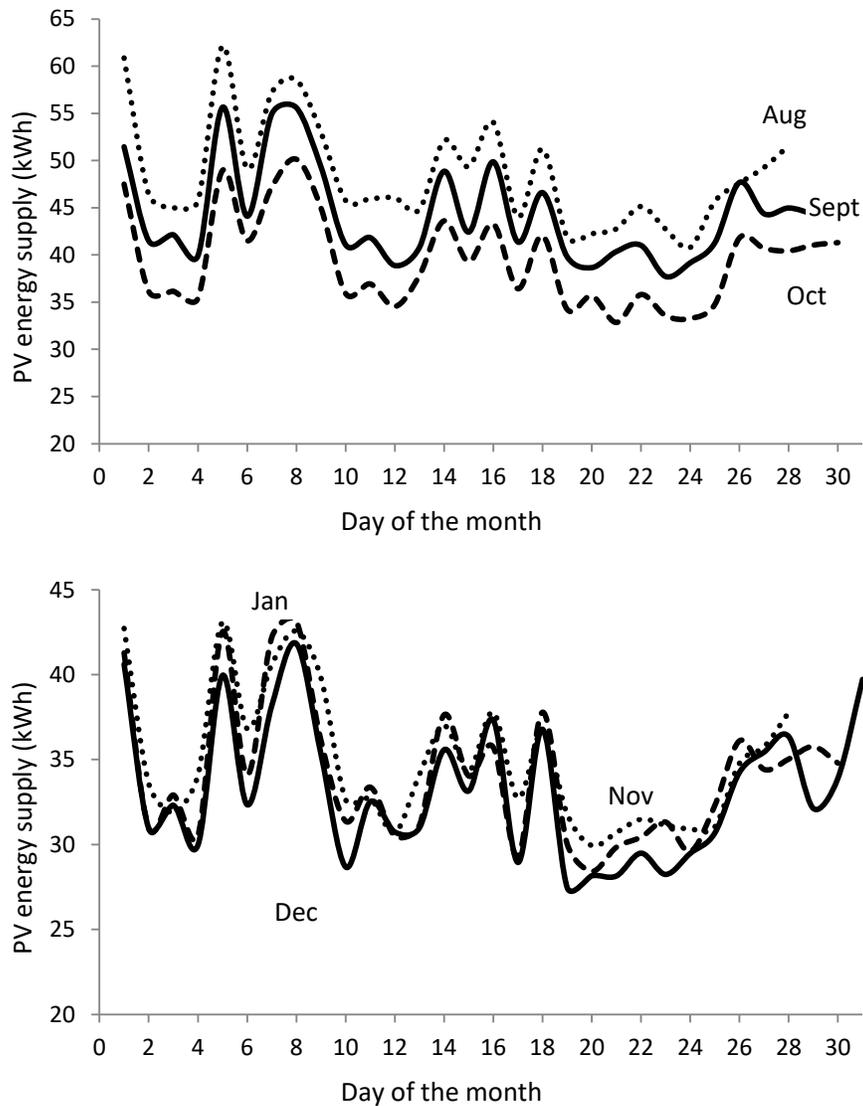


Figure 3: Daily evolution of the photovoltaic energy supply

Analyzing data from Figure 3, we observe that daily PV energy supply evolves from a minimum of 27.47 kWh to a maximum of 62.30 kWh. The PV evolution range varies depending on the season (Table 7).

| Month period | Feb-Apr | May-Jul | Aug-Oct | Nov-Jan |
|-----------------------------------|-------------|-------------|-------------|-------------|
| Season | Spring | Summer | Autumn | Winter |
| PV power supply range (kW) | 31.75-59.05 | 38.78-62.30 | 32.84-62.13 | 27.47-43.14 |
| Average value (kW) | 45.40 | 50.54 | 47.49 | 35.30 |
| Deviation over yearly average (%) | 8.49 | 20.78 | 13.48 | 15.64 |

Table 7: Seasonal average value and range for the PV energy supply

Figure 4 shows the monthly average value of the PV array energy supply. The dashed line corresponds to the yearly average value.

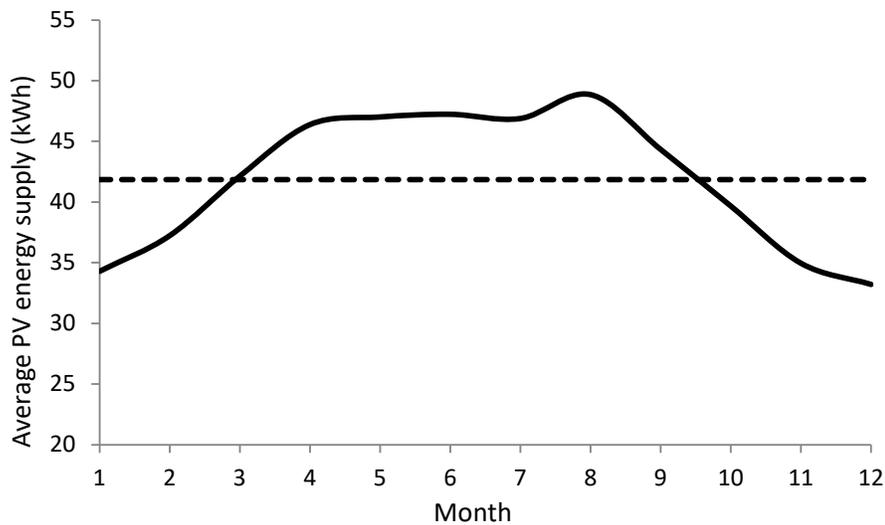


Figure 4: Monthly average value evolution of PV array energy supply

We notice that the monthly average PV energy supply evolution exceeds the yearly average value during the April through August period, and remains below the average from October to February. The second group of test results corresponds to the evolution of the household energy demand (Figure 5). Since sanitary water energy consumption depends on the geothermal heat pump supply, heating and air conditioning on the aérothermal unit, and electric consumption on the PV system, we divide the household energy demand into three groups, each corresponding to the above-mentioned energy consumptions.

Sanitary water is supplied by the health water tank, which requires thermal energy to regain the temperature level when hot water flows to the house. The power supply from the geothermal heat pump matches the energy demand corresponding to the hot water tank demand. We notice that daily thermal demand remained constant every day with a maximum deviation of 7.5%. On the other hand, the daily energy demand distribution also remained constant with a variation of less than 5%. Therefore, we can represent the geothermal heat pump power supply for a day, including daily deviation. Figure 5 shows the experimental results.

Analyzing the results from Figure 5, we notice that the geothermal heat pump supply depends on the season of the year, with a maximum in winter, and a minimum in summer, which agrees with expected values. The power supply operational range varies with the season of the year as shown in Table 8.

| Season | Maximum power (kW) | Minimum power (kW) |
|--------|--------------------|--------------------|
| Spring | 0.464 | 0.232 |
| Summer | 0.398 | 0.199 |
| Autumn | 0.571 | 0.285 |
| Winter | 0.690 | 0.345 |

Table 8: Maximum and minimum geothermal heat pump power supply for the different seasons of the year

We notice that the values shown in Table 6 fall within the operational range for the geothermal heat pump indicated in Table 5.

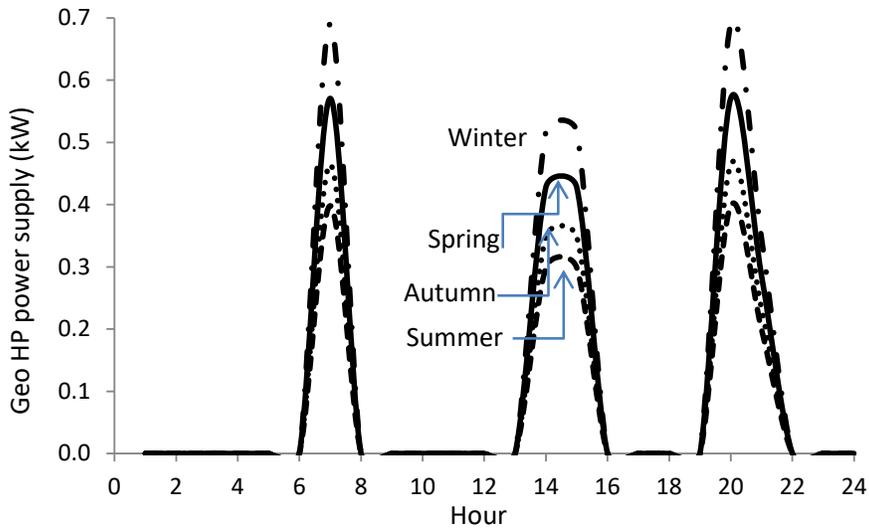
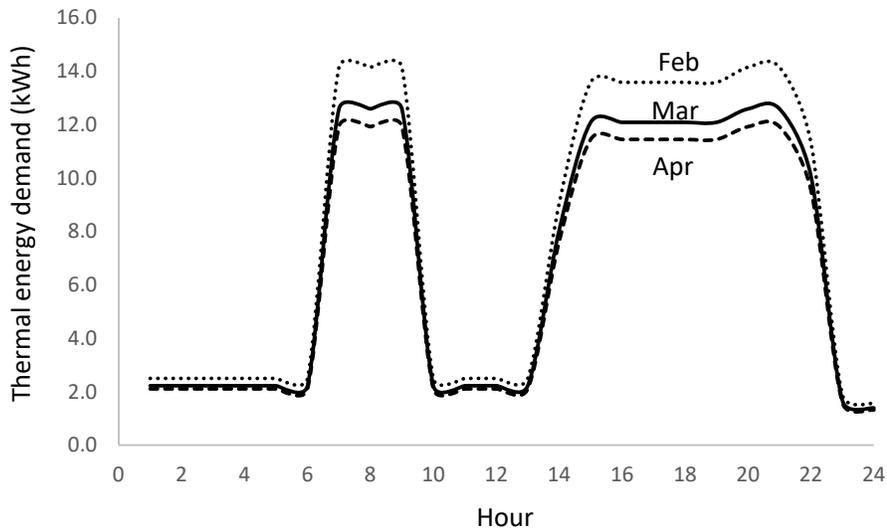


Figure 5: Geothermal Heat Pump daily power supply distribution

Integrating over the entire day, we obtain the geothermal heat pump daily energy consumption, obtaining a value of 2.76 kWh. Because the sanitary water use is regular, the heat pump energy consumption remains constant every day.

The third group of tests results corresponds to the evolution of the household thermal energy demand, heating and air conditioning (Figure 6). As in the case of the PV array energy supply, every figure corresponds to a group of three months, corresponding to a yearly period centered on the solar solstice and equinox; February to April for the spring equinox, May to July for the summer solstice, August to October for the autumn equinox, and November to January for the winter solstice.



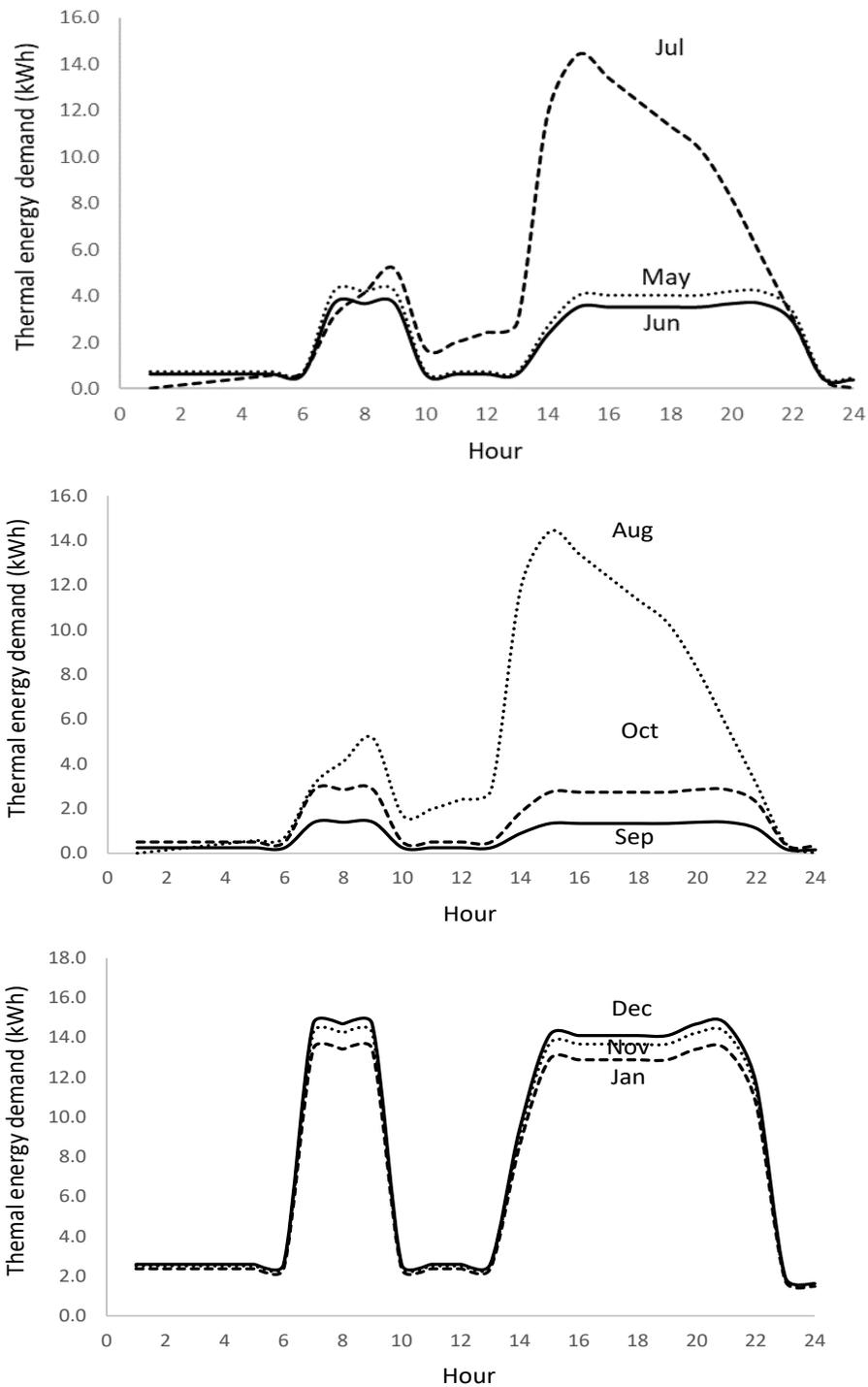


Figure 6: Hourly evolution of the household energy demand per month of the year

Heating operates from September to June, though in mild-temperature months, the heating energy demand is low. Nevertheless, we consider a heating contribution to the household energy demand during these months since the heating system is individual and not collective, meaning that the user may operate the heating system at will or automatically, generating the heating system activation if the inside temperature falls below the comfort level on any day during these months.

The analysis of household energy demand indicates that, from late autumn to early spring (November through April), the evolution is similar, with two peak daily periods: in the morning, 6 to 10 hours, and from early afternoon to near midnight, 14 to 22 hours. The rest of the day, energy demand is low due to the low or null human activity. On the other hand, in July and August, the energy demand

distribution is irregular, with a peak period near 3 p.m., and a slight slope from 3 p.m. to 10 p.m., due to air conditioning activation. The months of May, June, September, and October show a regular energy demand distribution, with slight differences between months. The energy demand in these months is much lower than in July and August because air conditioning consumes more energy than heating, and the operation time range is shorter.

Considering the monthly evolution of the energy consumption, we obtain (Figure 7):

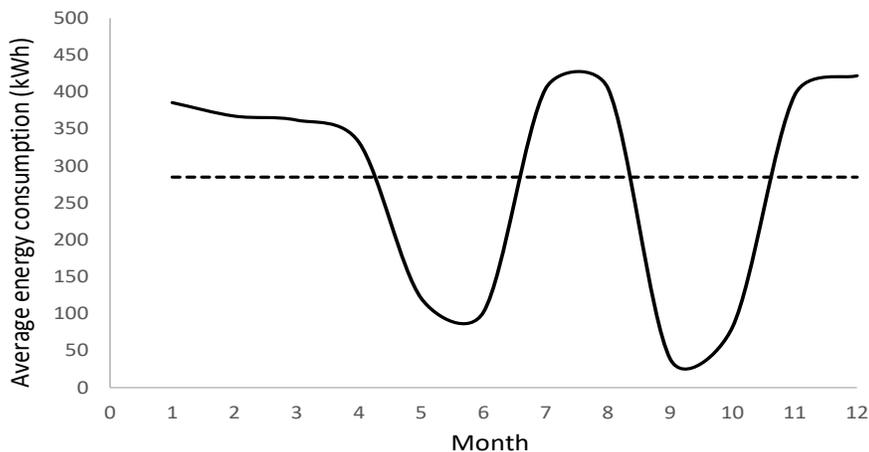


Figure 7: Monthly average value evolution of household energy demand (thermal)

We observe in Figure 7 that thermal energy demand evolves with the month of the year, following the climatic environment evolution, with maximum values in winter and summer, due to a higher demand in heating and air conditioning, and minimum values in May-June and September-October, corresponding to the mild climate in the operating zone.

Regarding the electrical energy consumption, we collected data from the control unit, obtaining the results shown in Figure 8. The data correspond to household appliances and accessories and do not include aérothermal or geothermal heat pumps, as their electric energy consumption corresponds with the previously analyzed thermal energy demand.

We only present daily hourly data, as the difference in daily electricity consumption throughout the year is negligible, less than 1%.

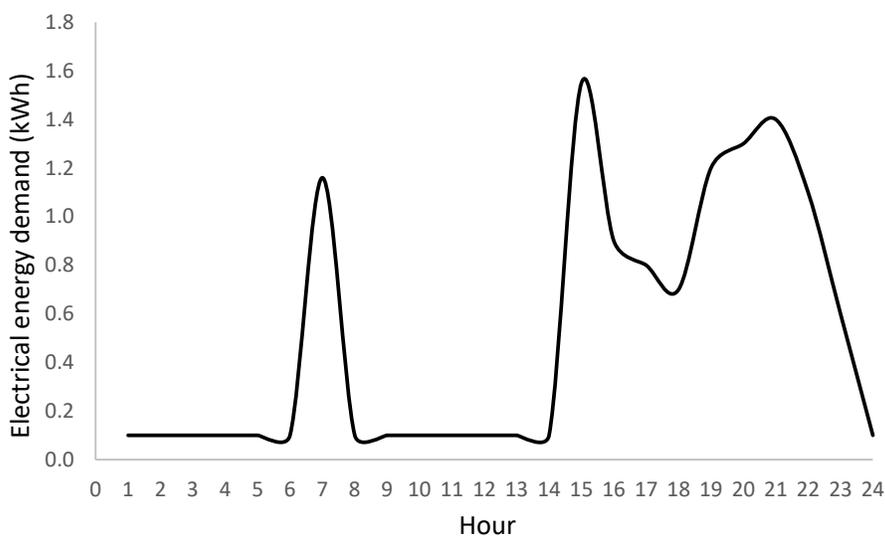


Figure 8: Daily hourly evolution of household energy demand (electrical)

The daily schedule corresponds to the specific habits of each family, with the highest levels of activity in the morning around 7:00 and from early afternoon, between 2:00 PM and midnight. This schedule may vary at any time if family habits change.

Once we have calculated all energy terms, we can determine the energy balance to establish the installation's sustainability. The battery bank is not included at this stage, as it only represents an energy storage system that supplies and receives energy from the installation to compensate for energy imbalances without affecting the overall energy balance. Equation 32 represents the overall energy balance.

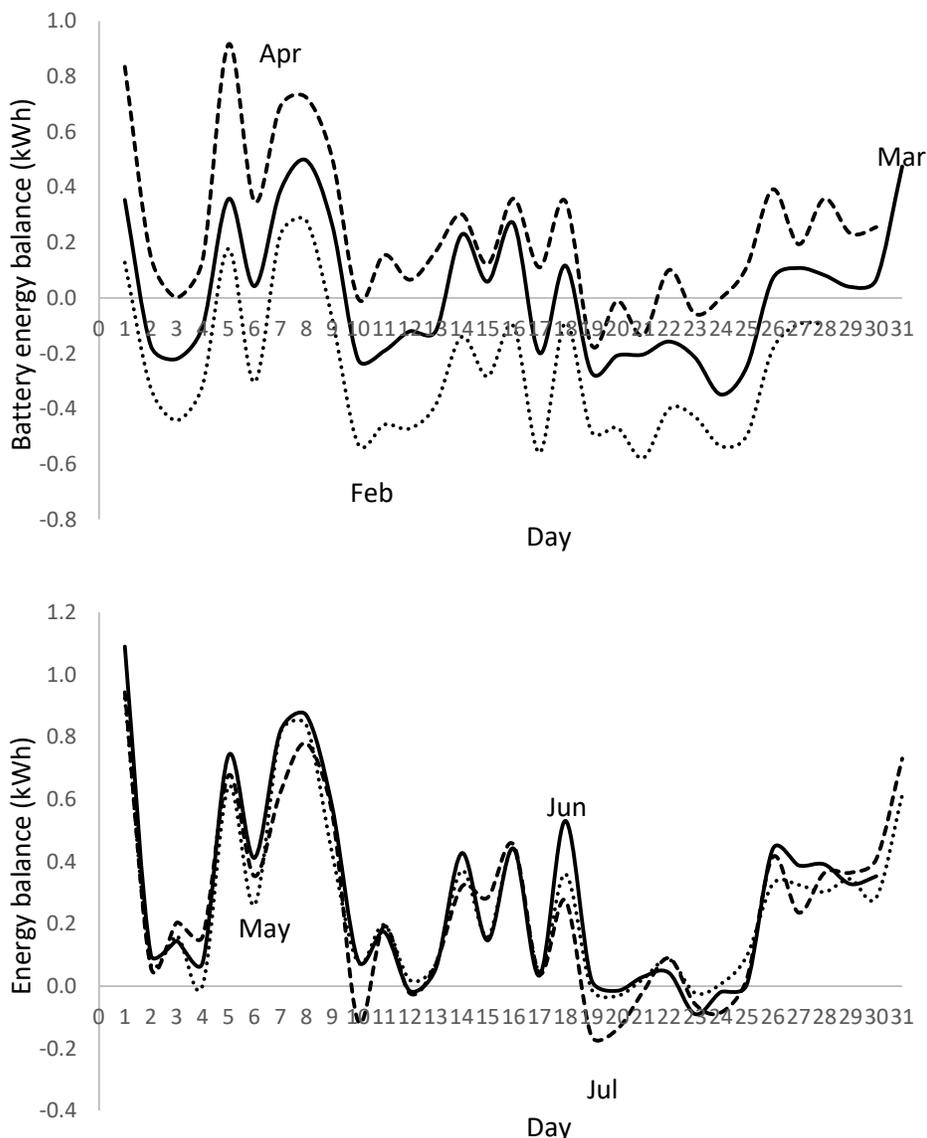
Computing all energy terms, we obtain:

$$\xi_{ov} = \sum_i (\xi_{PV})_i - \sum_j (\xi_{th})_j - \sum_k (\xi_{geo})_k - \sum_l (\xi_{el})_l \quad (32)$$

$$\xi_{ov} = 24330.53 - 18921.59 - 1007.04 - 4395.93 = 5.97 kWh \quad (33)$$

The result of the energy balance shows a negligible deviation, less than 0.02%, respect to the zero-energy balance. This value guarantees the system sustainability and the household energy self-sufficiency.

However, the overall annual energy balance is not entirely accurate, as it masks potential hourly or daily energy imbalances; therefore, it is advisable to analyze the evolution of the hourly and daily energy balances. Since the battery bank operates daily, the battery hourly energy balance is not suitable, and the daily energy balance is the optimal option. Figure 9 shows the experimental results.



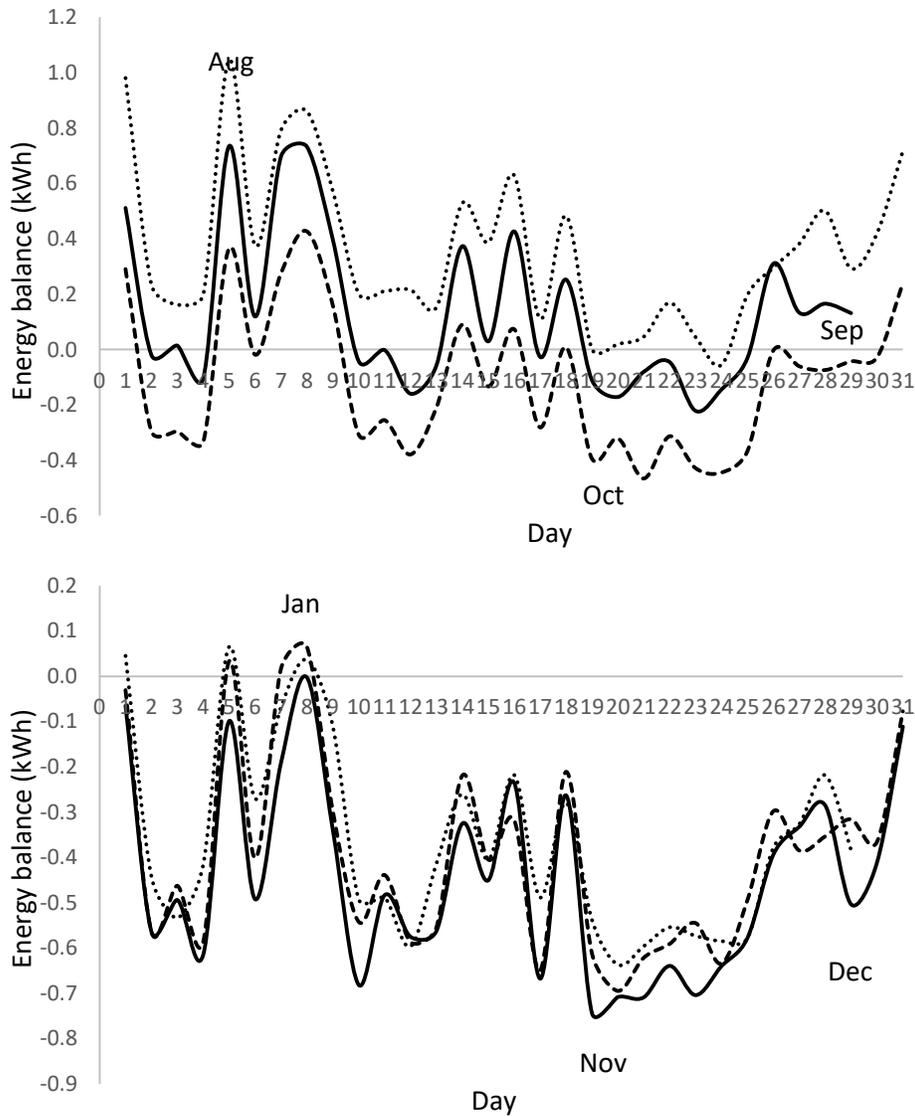


Figure 9: Battery daily energy balance

Applying values from Figure 9 to a battery block, the resulting capacity is (Table 9):

| | | | | | | |
|----------------|--------|--------|--------|--------|--------|--------|
| Month | Jan | Feb | Mar | Apr | May | Jun |
| Bat.cap. (kWh) | 21.527 | 16.189 | 10.810 | 5.066 | 0.990 | 2.692 |
| Month | Jul | Aug | Sep | Oct | Nov | Dec |
| Bat.cap. (kWh) | 4.946 | 1.729 | 6.650 | 14.446 | 19.094 | 23.044 |

Table 9: Monthly battery capacity to compensate energy imbalances

Operating with a battery block of 6.0 kWh, the number of elements for the battery block is:

$$N_{bat} = \frac{\max(\xi_{bat})}{C_{bat}} = \frac{23.044}{6.0} = 3.84 \rightarrow 4 \quad (34)$$

Matching the selected number for the battery block in the storage system design (Table 3).

5. Conclusions

A single-family home, whether detached or semi-detached, can be energy self-sufficient through the renewable energy systems use. The application of a hybrid system—a photovoltaic array and a geothermal unit—in this study validates this claim.

The sustainability of energy installations requires the use of highly efficient systems to supply thermal energy, such as high-performance air-source and geothermal heat pumps. Using these systems reduces the electrical energy to produce thermal energy, minimizing the size of the photovoltaic system and allowing the photovoltaic installation to meet the house's energy needs.

A home that uses sustainable energy requires a battery bank with sufficient capacity to compensate for energy imbalances throughout the year, resulting from the variability in the photovoltaic system power supply and the variation in energy demand due to annual weather conditions variability.

Designing an energy-efficient system compatible with a home's self-consumption requires sufficient space for installing the photovoltaic system, the air source heat pump, and the geothermal heat pump with its heat exchanger. This configuration is feasible in single-family homes and semi-detached houses with a small adjacent plot of land and a terrace or rooftop with enough space for the photovoltaic system.

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